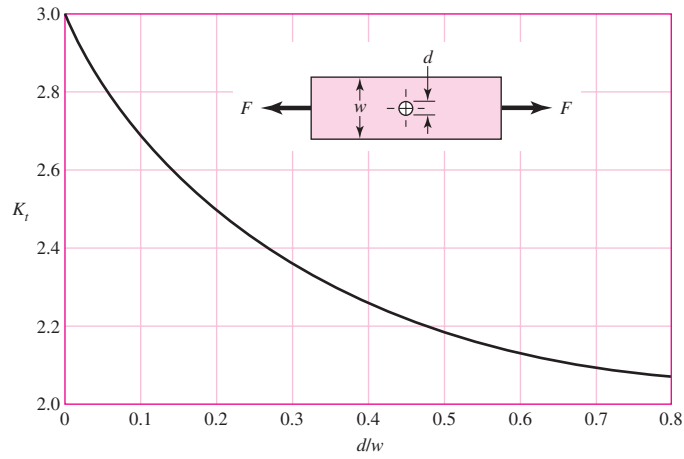


**Table A-15**

Charts of Theoretical Stress-Concentration Factors  $K_t^*$

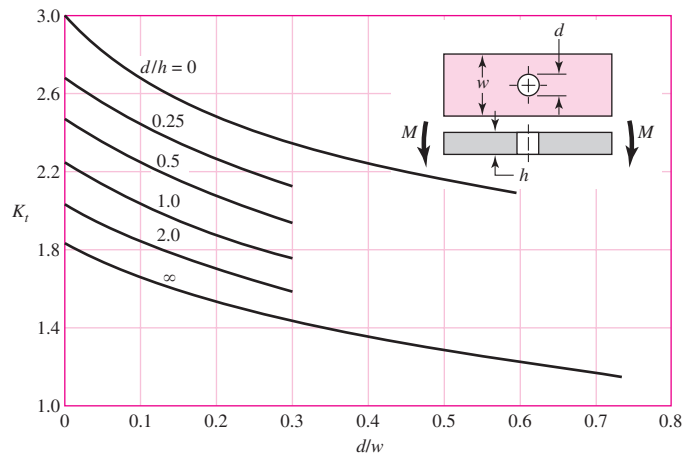
**Figure A-15-1**

Bar in tension or simple compression with a transverse hole.  $\sigma_0 = F/A$ , where  $A = (w - d)t$  and  $t$  is the thickness.



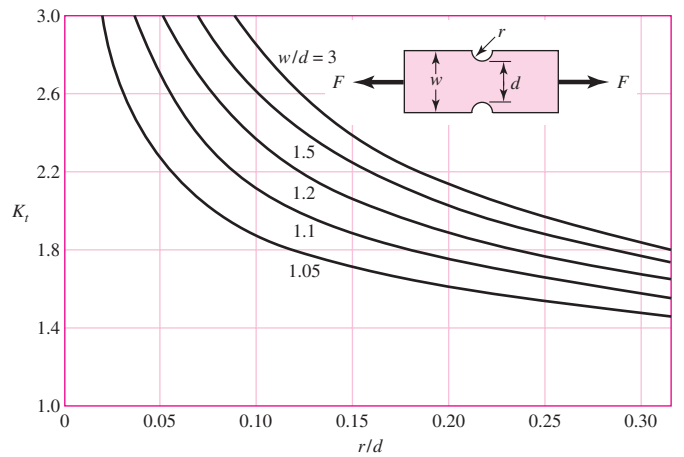
**Figure A-15-2**

Rectangular bar with a transverse hole in bending.  $\sigma_0 = Mc/I$ , where  $I = (w - d)h^3/12$ .



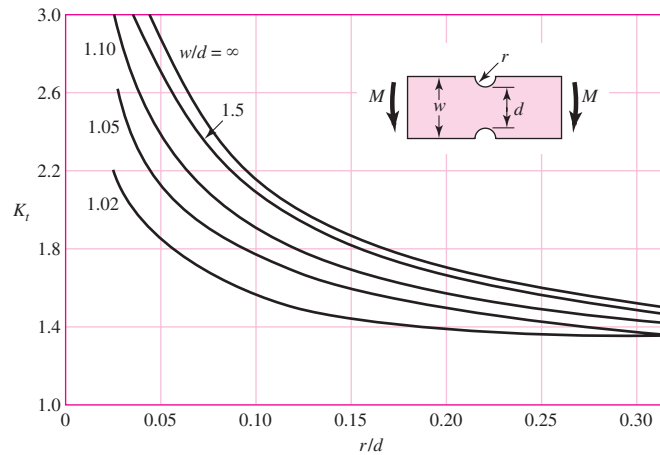
**Figure A-15-3**

Notched rectangular bar in tension or simple compression.  $\sigma_0 = F/A$ , where  $A = dt$  and  $t$  is the thickness.

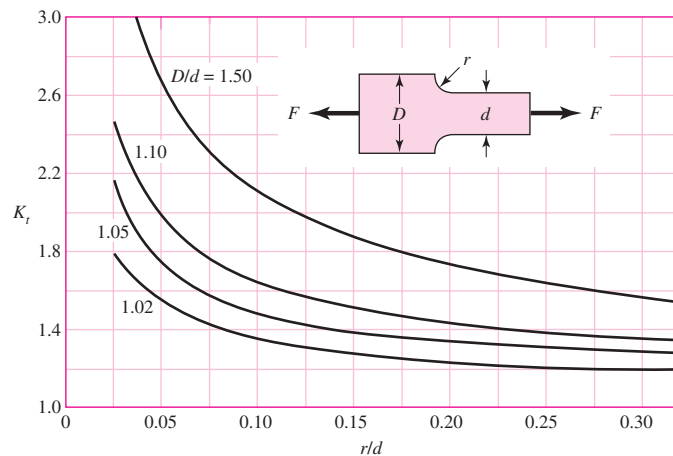


**Table A-15**Charts of Theoretical Stress-Concentration Factors  $K_t^*$  (Continued)**Figure A-15-4**

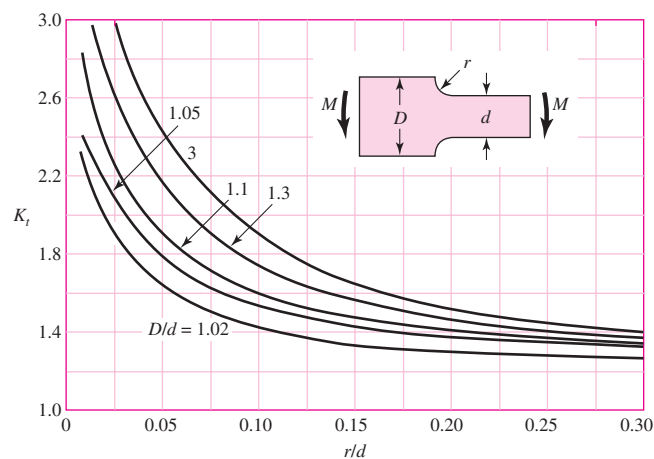
Notched rectangular bar in bending.  $\sigma_0 = Mc/I$ , where  $c = d/2$ ,  $I = td^3/12$ , and  $t$  is the thickness.

**Figure A-15-5**

Rectangular filleted bar in tension or simple compression.  $\sigma_0 = F/A$ , where  $A = dt$  and  $t$  is the thickness.

**Figure A-15-6**

Rectangular filleted bar in bending.  $\sigma_0 = Mc/I$ , where  $c = d/2$ ,  $I = td^3/12$ ,  $t$  is the thickness.



(continued)

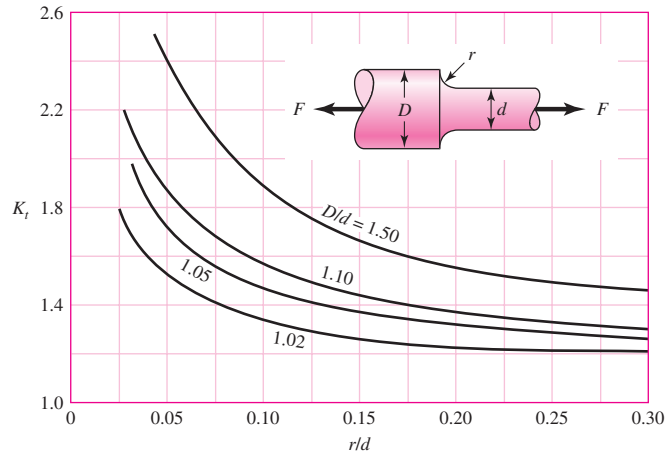
\*Factors from R. E. Peterson, "Design Factors for Stress Concentration," Machine Design, vol. 23, no. 2, February 1951, p. 169; no. 3, March 1951, p. 161, no. 5, May 1951, p. 159; no. 6, June 1951, p. 173; no. 7, July 1951, p. 155. Reprinted with permission from Machine Design, a Penton Media Inc. publication.

**Table A-15**

Charts of Theoretical Stress-Concentration Factors  $K_t^*$  (Continued)

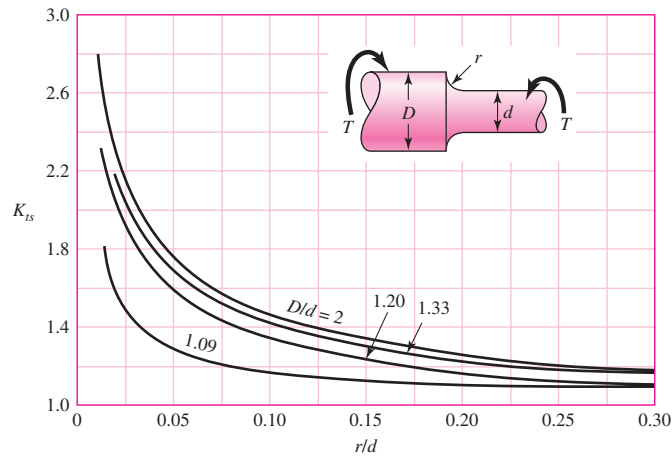
**Figure A-15-7**

Round shaft with shoulder fillet in tension.  $\sigma_0 = F/A$ , where  $A = \pi d^2/4$ .



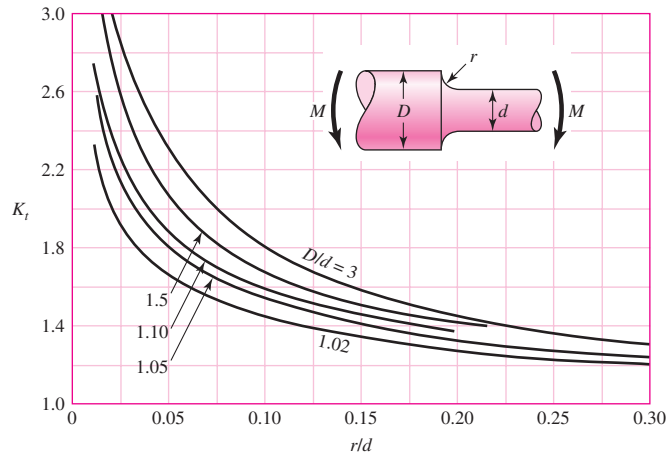
**Figure A-15-8**

Round shaft with shoulder fillet in torsion.  $\tau_0 = Tc/J$ , where  $c = d/2$  and  $J = \pi d^4/32$ .



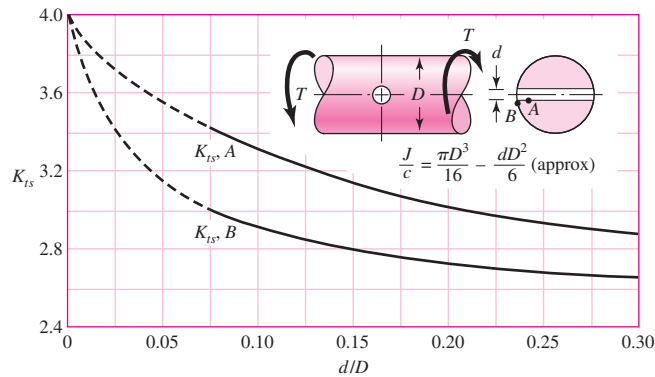
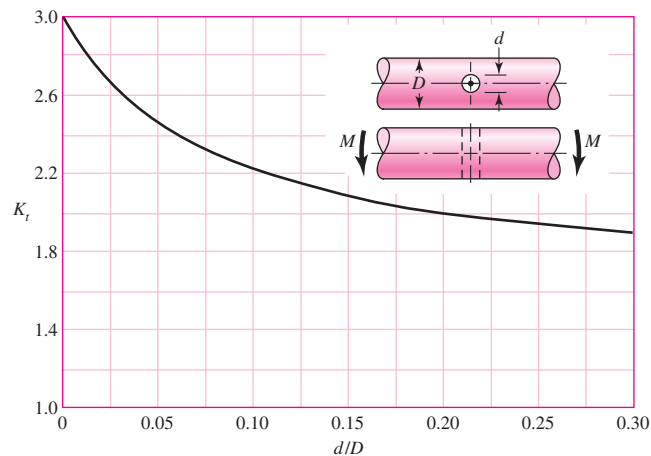
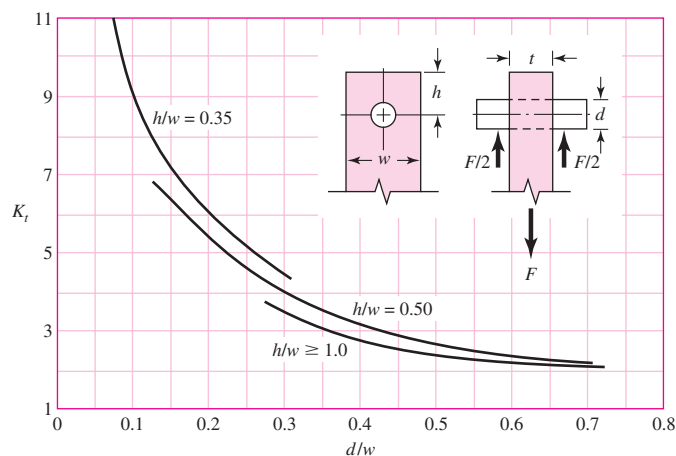
**Figure A-15-9**

Round shaft with shoulder fillet in bending.  $\sigma_0 = Mc/I$ , where  $c = d/2$  and  $I = \pi d^4/64$ .



**Table A-15**Charts of Theoretical Stress-Concentration Factors  $K_t^*$  (Continued)**Figure A-15-10**

Round shaft in torsion with transverse hole.

**Figure A-15-11**Round shaft in bending with a transverse hole.  $\sigma_0 = M/[(\pi D^3/32) - (dD^2/6)]$ , approximately.**Figure A-15-12**Plate loaded in tension by a pin through a hole.  $\sigma_0 = F/A$ , where  $A = (w - d)t$ . When clearance exists, increase  $K_t$  35 to 50 percent. (M. M. Frocht and H. N. Hill, "Stress-Concentration Factors around a Central Circular Hole in a Plate Loaded through a Hole in Hole," *J. Appl. Mechanics*, vol. 7, no. 1, March 1940, p. A-5.)

(continued)

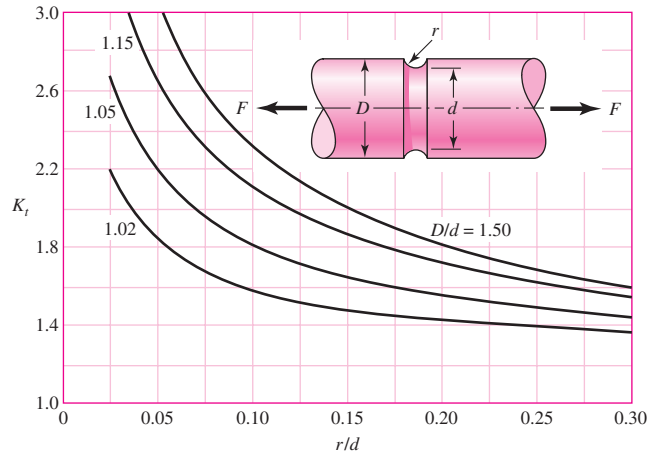
\*Factors from R. E. Peterson, "Design Factors for Stress Concentration," *Machine Design*, vol. 23, no. 2, February 1951, p. 169; no. 3, March 1951, p. 161, no. 5, May 1951, p. 159; no. 6, June 1951, p. 173; no. 7, July 1951, p. 155. Reprinted with permission from Machine Design, a Penton Media Inc. publication.

**Table A-15**

Charts of Theoretical Stress-Concentration Factors  $K_t^*$  (Continued)

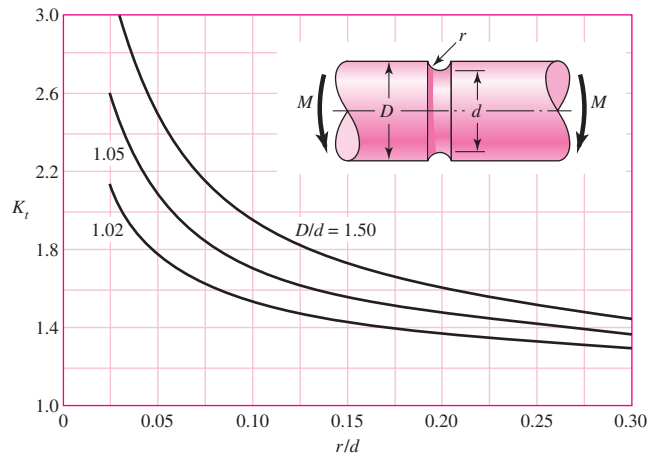
**Figure A-15-13**

Grooved round bar in tension.  
 $\sigma_0 = F/A$ , where  $A = \pi d^2/4$ .



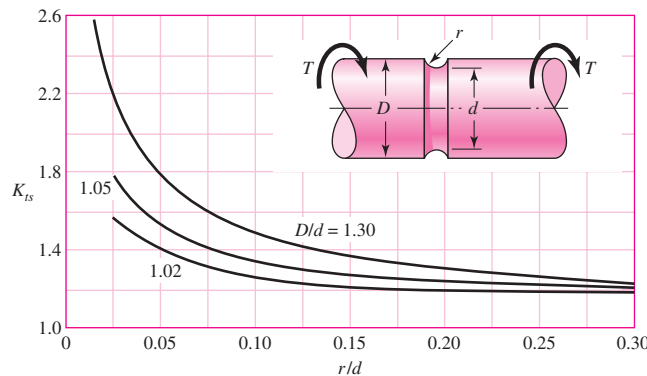
**Figure A-15-14**

Grooved round bar in bending.  
 $\sigma_0 = Mc/I$ , where  $c = d/2$   
 and  $I = \pi d^4/64$ .



**Figure A-15-15**

Grooved round bar in torsion.  
 $\tau_0 = Tc/J$ , where  $c = d/2$  and  
 $J = \pi d^4/32$ .



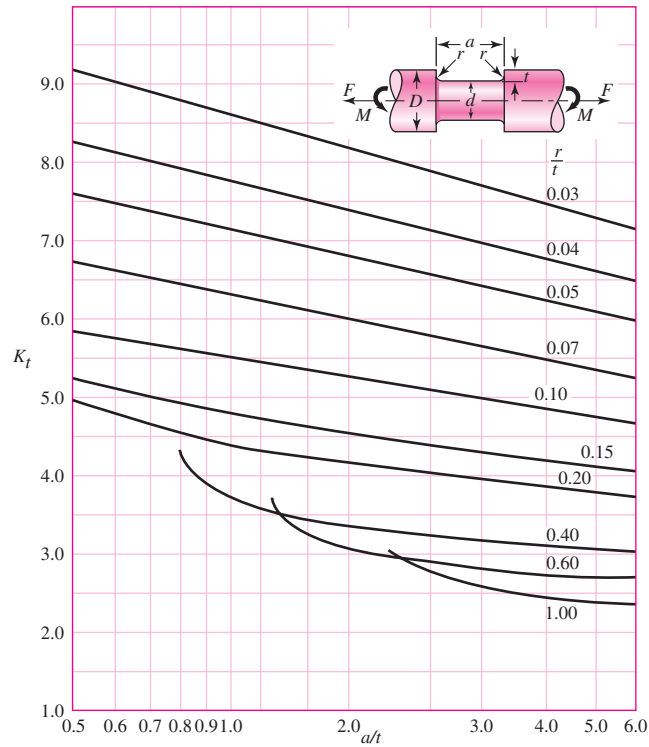
\*Factors from R. E. Peterson, "Design Factors for Stress Concentration," Machine Design, vol. 23, no. 2, February 1951, p. 169; no. 3, March 1951, p. 161, no. 5, May 1951, p. 159; no. 6, June 1951, p. 173; no. 7, July 1951, p. 155. Reprinted with permission from Machine Design, a Penton Media Inc. publication.

**Table A-15**Charts of Theoretical Stress-Concentration Factors  $K_t^*$  (Continued)**Figure A-15-16**

Round shaft with flat-bottom groove in bending and/or tension.

$$\sigma_0 = \frac{4F}{\pi d^2} + \frac{32M}{\pi d^3}$$

Source: W. D. Pilkey, *Peterson's Stress-Concentration Factors*, 2nd ed. John Wiley & Sons, New York, 1997, p. 115.



(continued)

**Table A-15**

Charts of Theoretical Stress-Concentration Factors  $K_t^*$  (Continued)

**Figure A-15-17**

Round shaft with flat-bottom groove in torsion.

$$\tau_0 = \frac{16T}{\pi d^3}$$

Source: W. D. Pilkey, *Peterson's Stress-Concentration Factors*, 2nd ed. John Wiley & Sons, New York, 1997, p. 133

