

# Welcome to the PMG Educational Program

Sponsored by:



# *An Overview on ACCA's Residential HVAC System Design Process*

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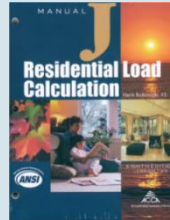
Cedar Springs, MI

# Presentation Overview:

## *ACCA Manual J, Manual S, Manual D*



1. Provide a fundamental understanding on the basics of what it takes to do an accurate residential mechanical system design:



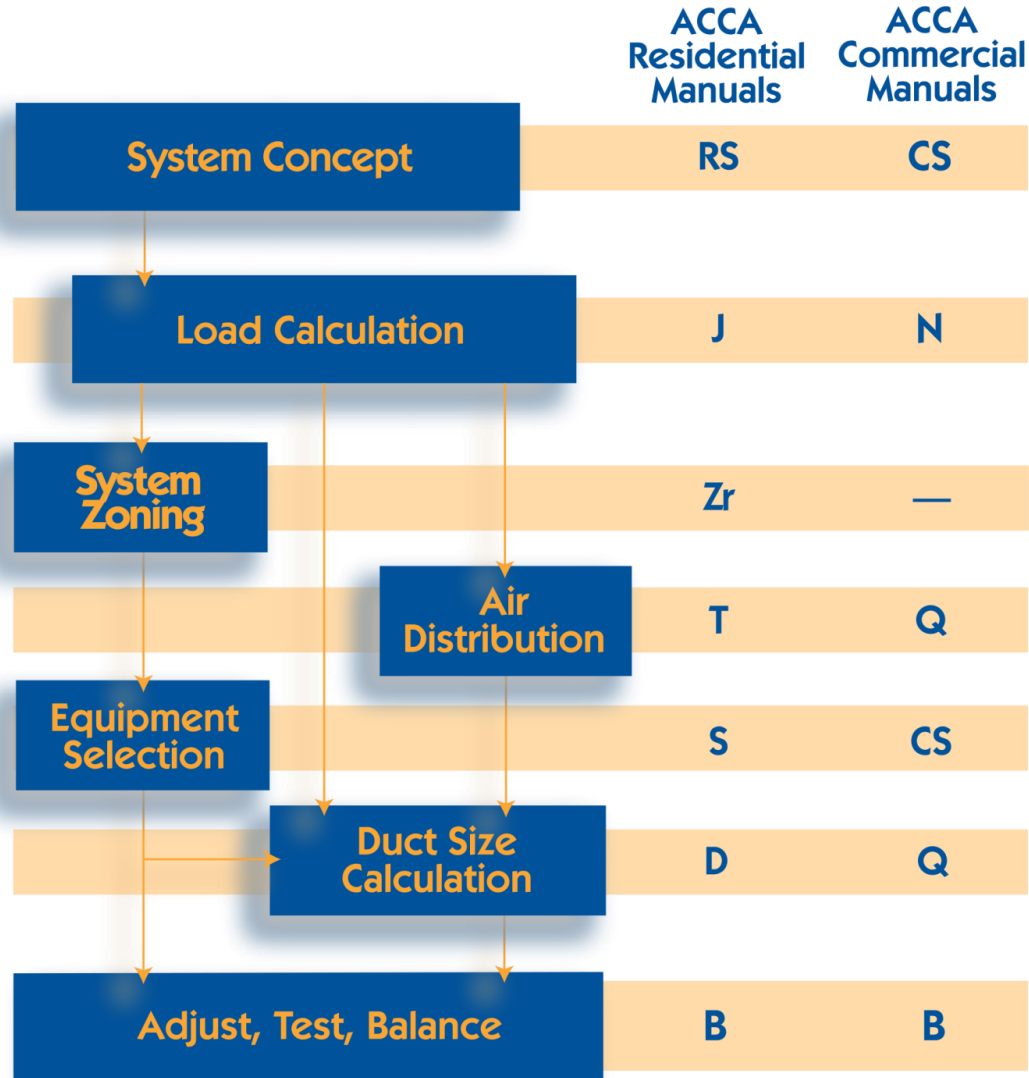
2. Provide verification points and caveats
  - Code officials: For the purpose of issuing a permit
  - Quality control personnel: Checking consistency/accuracy
3. Highlight relevant ACCA resources and opportunities

# Designer's Objective

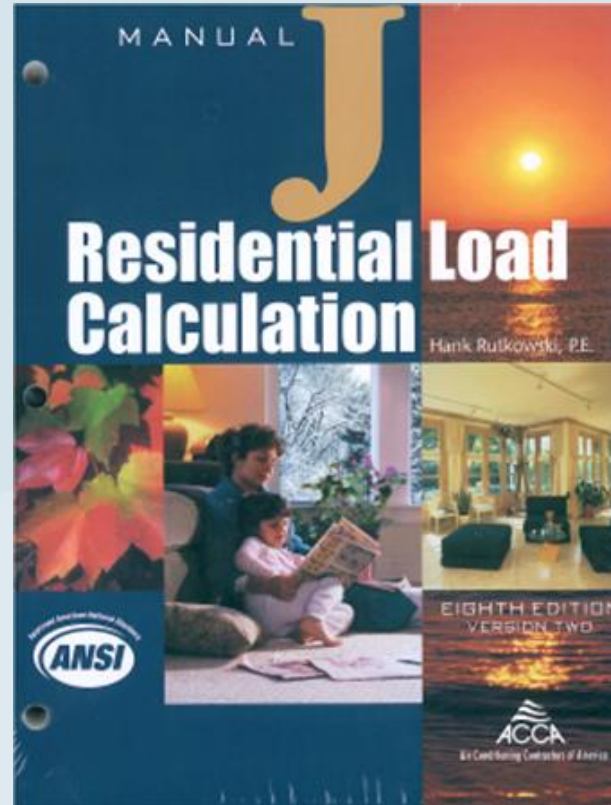
To design a mechanical system that can add (heating) or remove (cooling) heat energy at a rate (BTUs per hour) that will allow the home's indoor environment to achieve the design conditions.

This will keep occupants comfortable and safe and provide for energy-efficient operation.

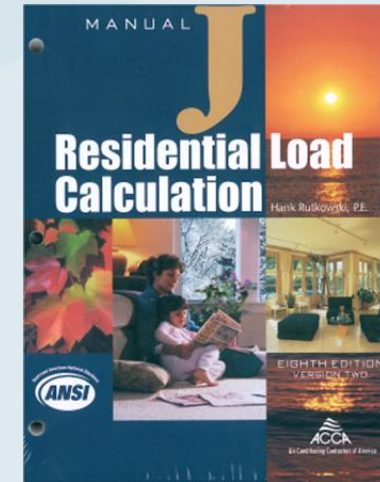
# System Design Process



# Part 1 – Load Calculation



- **Standard required in:**
  - 2015 IRC §M1401.3, and
  - 2015 IECC §R403.7
- **Comprised of two sections**
  - Normative: 9 pages of text and 200 pages of tabular information that are the enforceable requirements
  - Informative: 390 pages of in depth discussion, documentation, and examples
- **Latest ANSI approval in Feb 2016**



## Summer

- Heat flows INTO the home
  - Sensible heat – dry heat (dry bulb; thermometer)
  - Latent heat – wet heat (wet bulb; humidity)

*Heat Gain ... so we need cooling*

## Winter

- Heat flows OUT of the home
  - Sensible heat only

*Heat Loss ... so we need heating*

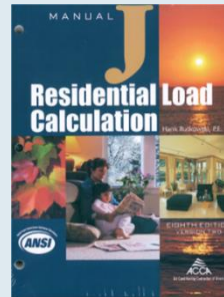
Heat flow is a rate; the units are Btu/h.  
(Analogous to mph).



# Manual J Load Design Conditions

Two design conditions ... hence, two sets of peak loads.

	Outdoor Design Temp (Geographic-specific)	Indoor Design Temp
Heat Gain (summer)	1% db condition	75 F
Heat Loss (winter)	99% db condition	70 F



# Loads That Must Be Accounted For

(as applicable to the specific home)



- **Fenestration** (windows, glass doors, skylights)
- **Opaque panels** (wood/metal doors, above & below grade walls, partition walls, ceilings, floors)
- **Infiltration**
- **Ventilation**
- **Internal** (number of people and appliances)
- **System** (ducts and blower)

# Basic Load Equation

$$\text{Load} = U \times A \times \Delta T$$

**U** = the heat transfer performance index  
(how well a material transfers heat; it's the reciprocal of R-value)

**A** = the Area of the surface (window, wall, ceiling, etc.)

**$\Delta T$**  = the temperature difference across the surface

**Load** units are Btu/h

## Simple load calculation – MJ8<sub>AE</sub> (Abridged Edition)

- Dwelling must be 100% compatible with AE Checklist
- Can be done by hand or using ACCA MJ8 speedsheet

## Full load calculation – Full MJ8

- Can be done by hand, but extremely time consuming
- Usually use third party software<sup>1</sup>

<sup>1</sup> ACCA vets third party software for compliance with MJ8 procedures, those that pass received “Powered by Manual J” recognition (see: <http://www.acca.org/standards/approved-software>)



# Load Calculation

## Min. Verification Points



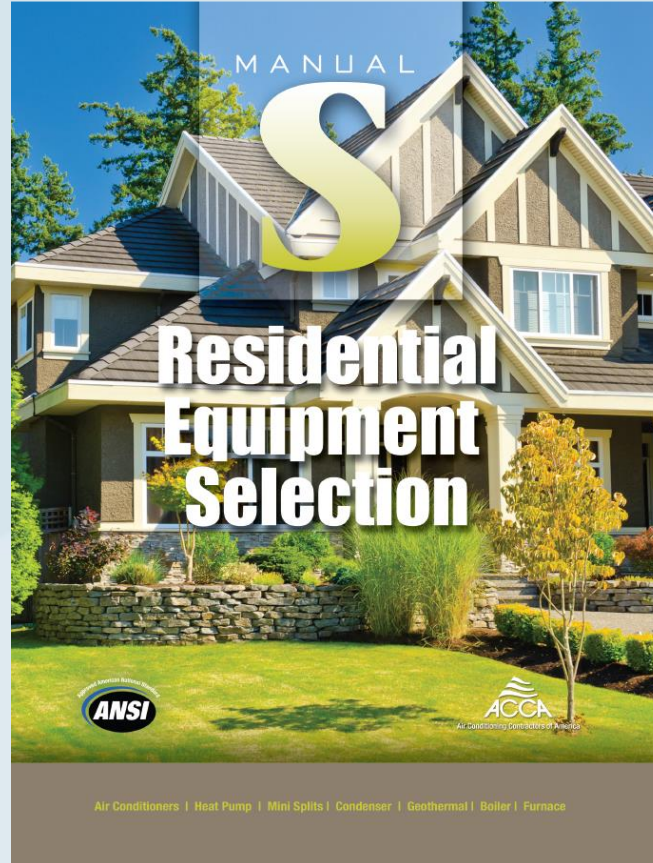
- **Location** (City, State)
- **Outdoor design temperatures and grains** (Why deviating from MJ8 Tables 1A or 1B?)
- **Indoor design temperatures** (75°F db cooling, 70°F db heating unless superseded by code/regulation)
- **Orientation** matches actual home or plan
- **Occupants** = number of bedrooms + 1
- Conditioned **floor area** = home or plan
- **Eave overhang** depth and **internal shading** = home or plan / default
- Number of **skylights** = home or plan
- **Sensible + latent heat gain = total heat gain**

## Some practitioners will try to fudge the numbers to get bigger loads:

- Change the design temperatures (outdoor and/or indoor)
- Design to the worst case scenario (e.g., very loose house)
- Add more occupants than 'number of bedrooms plus 1'
- Calculate duct loads even when ducts in conditioned space
- Not include window overhangs and shading
- Puff up internal loads
- Use a factor of safety

The above practices are not supported by ACCA. Manual J instructs practitioners to be thorough and reflect the ACTUAL conditions.

# Part 2 - Equipment Selection





# ANSI/ACCA 3 Manual S - 2014



- **Standard required in:**
  - 2015 IRC §M1401.3, and
  - 2015 IECC §R403.7
- **Comprised of two sections:**
  - Normative: 22 pages of enforceable requirements
  - Informative: 270 pages of in-depth discussion, documentation, and examples
- **Latest ANSI approval in May 2014**



## 1. Start with sizing values

- MJ8 heating load: For furnaces and boilers
- MJ8 cooling load: For cooling-only and heat pump units

## 2. Manual S provides sizing rules

- Sets upper and lower limits for equipment total capacity

## 3. Designer must use OEM performance data

- Capacity values must be for operating conditions

# Size Limits For Each Equipment Type

Size Limits for Cooling-Only Equipment			
Equipment Type	Single Speed	Two Speed	Variable Speed See Note 8
	Ducted or Ductless Total Cooling Capacity		
Air-Air	Max = 1.15 Min = 0.90	Max = 1.20 Min = 0.90 FS	Max = 1.30 Min = 0.90 RS
Water-Air pipe loop system	Max = 1.15 Min = 0.90	Max = 1.20 Min = 0.90 FS	Max = 1.30 Min = 0.90 RS
Water-Air open-piping system	Max = 1.25 Min = 0.90	Max = 1.30 Min = 0.90 FS	Max = 1.35 Min = 0.90 RS
Zone Damper Systems	To minimize excess air issues, zone damper systems shall have as little excess cooling capacity as possible when full-cooling capacity is compared to the <b>Manual J</b> block load for the space served.		

Size Limits for Fossil Fuel Furnaces			
Output Capacity for Heating-Only	Single Stage	Multi Stage	Modulate Burner
		Sizing value to 1.4 x sizing value	Sizing value to 1.4 x sizing value at full capacity
Preferred <sup>3</sup> Output Capacity for Heating and Cooling	Sizing value to 1.4 x sizing value	Sizing value to 1.4 x sizing value at full capacity	Sizing value to 1.4 x sizing value at full capacity
Maximum <sup>4</sup> Output Capacity for Heating and Cooling	Sizing value to 2.0 x sizing value	Sizing value to 2.0 x sizing value at full capacity	Sizing value to 2.0 x sizing value at full capacity
Zone Damper Systems	Zone damper systems should have as little excess capacity as possible when full capacity is compared to the <b>Manual J</b> block load for the space served.		

# Heat Pump Sizing Limits

Size Limits for Condition A Heat Pumps JSHR < 0.95; or HDD / CDD < 2.0				
Equipment Type	Single Speed	Two Speed	Variable Speed	
	Ducted or Ductless		Ducted	Ductless
Air-Air	Max = 1.15 Min = 0.90	Max = 1.20 Min = 0.90 FS	Max = 1.20 Min = 0.90 RS	Max = 1.30 Min = 0.90 RS
Water-Air pipe loop system	Max = 1.15 Min = 0.90	Max = 1.20 Min = 0.90 FS	Max = 1.20 Min = 0.90 RS	
Water-Air open pipe system	Max = 1.25 Min = 0.90	Max = 1.25 Min = 0.90 FS	Max = 1.25 Min = 0.90 RS	

Size Limits for Condition B Heat Pumps JSHR = 0.95 or greater; and HDD / CDD = 2.0 or greater			
Equipment Type	Single Speed	Two Speed	Variable Speed
Air-Air Ducted or Ductless	Max = +15,000 Min = 0.90	Max = +15,000 Min = 0.90 FS	Max = +15,000 Min = 0.90 RS
Water-Air pipe loop system	Max = +15,000 Min = 0.90	Max = +15,000 Min = 0.90 FS	Max = +15,000 Min = 0.90 RS
Water-Air open pipe system	Max = +15,000 Min = 0.90	Max = +15,000 Min = 0.90 FS	Max = +15,000 Min = 0.90 RS

Designer must heed the notes for the tables.

A piece of equipment's AHRI rating is evaluated for air at:  
80°F db / 67°F wb entering the indoor unit, and  
95°F db entering the outdoor unit.

A standardized testing point for equipment capacity and efficiency, but inappropriate for use in equipment sizing and selection.

**No one wants an 80°F indoor environment in the summer!**  
And not every location will have a 95°F outdoor design temperature.

# Equipment Sizing / Selection

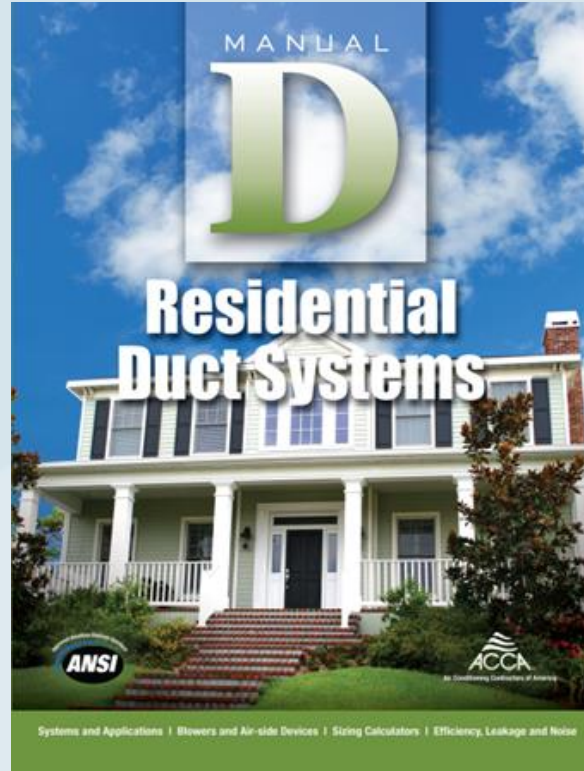
## Min. Verification Points

	Cooling Equipment	Heating Equipment
<i>Equipment Information</i>	<ul style="list-style-type: none"><li>• Type</li><li>• Model</li></ul>	<ul style="list-style-type: none"><li>• Type</li><li>• Model</li></ul>
<i>Capacities satisfy design conditions</i>	<ul style="list-style-type: none"><li>• Sensible Capacity</li><li>• Latent Capacity</li><li>• Total Capacity</li></ul>	<ul style="list-style-type: none"><li>• Total Output Capacity</li><li>• Auxiliary Heating Cap.</li></ul>
<i>Within load sizing limits</i>	<ul style="list-style-type: none"><li>• To be verified</li></ul>	<ul style="list-style-type: none"><li>• To be verified</li></ul>
<i>Blower Info (at design conditions)</i>	<ul style="list-style-type: none"><li>• CFM</li><li>• ESP</li></ul>	<ul style="list-style-type: none"><li>• CFM</li><li>• ESP</li></ul>

## Some designers will:

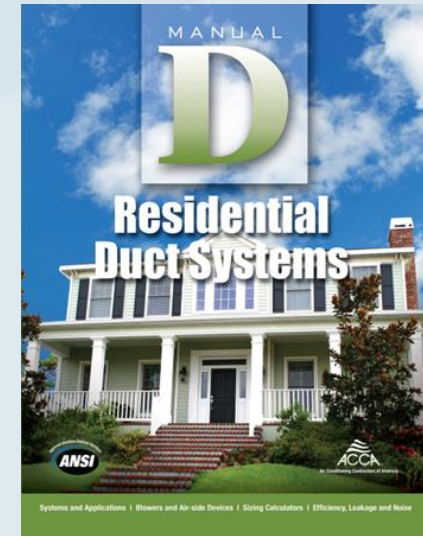
- Seek (incorrectly) to use AHRI rated capacities instead of OEM engineering performance data
- Not interpolating the OEM performance data for the capacity at design conditions
- Misread / misapply OEM performance data tables  
(can be very confusing, and will come in different configurations)
- Round up to next size
- Push for equipment outside of the sizing limits

# Part 3 – Duct System Design





- **Standard required in:**
  - 2015 IRC §M1601.1 and §M1602.2
  - 2015 IMC §603.2
- **Comprised of two sections**
  - Normative: 43 pages of enforceable requirements
  - Informative: 213 pages of in-depth discussion, documentation, and examples
- Latest ANSI Approval in Oct 2016



# Friction Rate Worksheet

## Friction Rate Worksheet

### Step 1) Manufacturer's Blower Data

External static pressure (ESP) = \_\_\_\_\_ IWC    Cfm = \_\_\_\_\_

### Step 2) Component Pressure Losses (CPL)

Direct expansion refrigerant coil \_\_\_\_\_  
 Electric resistance heating coil \_\_\_\_\_  
 Hot water coil \_\_\_\_\_  
 Heat exchanger \_\_\_\_\_  
 Low efficiency filter \_\_\_\_\_  
 High or mid-efficiency filter \_\_\_\_\_  
 Electronic filter \_\_\_\_\_  
 Other items that impede airflow \_\_\_\_\_  
 Supply outlet \_\_\_\_\_  
 Return grille \_\_\_\_\_  
 Balancing damper \_\_\_\_\_  
 Zone damper (full open) \_\_\_\_\_

Total component losses (CPL) \_\_\_\_\_ IWC

### Step 3) Available Static Pressure (ASP)

ASP = (ESP - CPL) = ( \_\_\_\_\_ - \_\_\_\_\_ ) = \_\_\_\_\_ IWC

### Step 4) Total Effective Length (TEL)

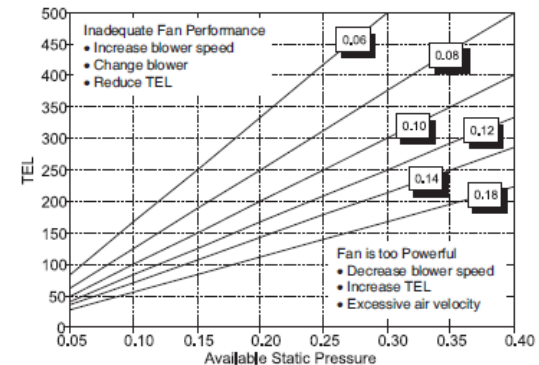
Supply-side TEL + Return-side TEL = ( \_\_\_\_\_ + \_\_\_\_\_ ) = \_\_\_\_\_ Feet

### Step 5) Friction Rate Design Value (FR)

FR value from friction rate chart = \_\_\_\_\_ IWC/100 Ft

$$FR = \frac{ASP \times 100}{TEL}$$

Friction Rate Chart



### Step 1) Manufacturer's Blower Data

External static pressure (ESP) = 0.67 IWC      Cfm = 1,000

### Step 2) Component Pressure Losses (CPL)

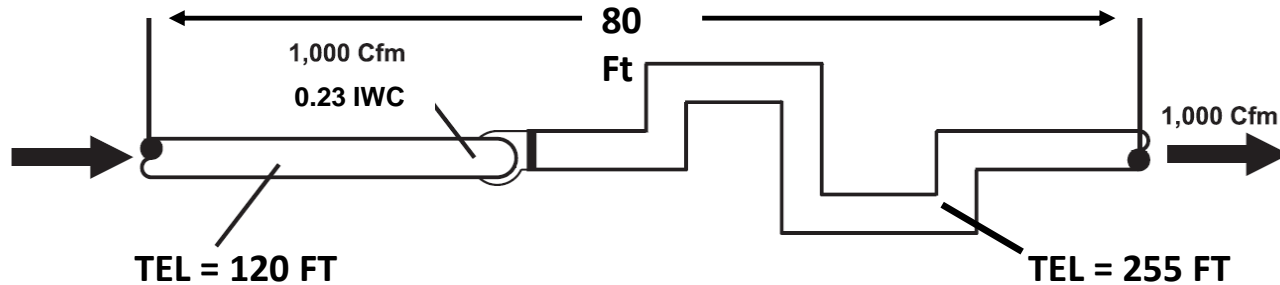
Direct expansion refrigerant coil	0.25
Electric resistance heating coil	_____
Hot water coil	_____
Heat exchanger	_____
Low efficiency filter	_____
High or mid-efficiency filter	_____
Electronic filter	0.10
Other items that impede airflow	_____
Supply outlet	0.03
Return grille	0.03
Balancing damper	0.03
Zone damper (full open)	_____
Total component losses (CPL)	0.44 IWC

### Step 3) Available Static Pressure (ASP)

ASP = (ESP - CPL) = ( 0.67 - 0.44 ) = 0.23 IWC

## Step 4) Total Effective Length (TEL)

$$\text{Supply-Side TEL} + \text{Return-Side TEL} = ( 255 + 120 ) = 375 \text{ Feet}$$



<p>R/D</p> <p>Miterd (R =</p> <p>0.75</p> <p>1.0</p> <p>1.5 or Larger</p>	<p>4A</p> <p>EL = 30</p>	<p>4B</p> <p>EL = 35</p>	<p>4C</p> <p>EL = 60</p>	<p>4D</p> <p>EL = 55</p>	<p>4E</p> <p>EL = 70</p>
	<p>4F</p> <p>EL = 45</p>	<p>4G</p> <p>EL = 80</p>	<p>4H</p> <p>EL = 50</p>	<p>4I</p> <p>EL = 10</p>	<p>4J</p> <p>EL = 30</p>
	<p>4K</p> <p>EL = 30</p>	<p>4L</p> <p>EL = 80</p>	<p>4M</p> <p>EL = 20</p>	<p>4N</p> <p>EL = 45</p>	<p>4O</p> <p>EL = 20</p>

## Step 5) Friction Rate Design Value (FR)

FR value from friction rate chart = 0.06 IWC/100 Ft

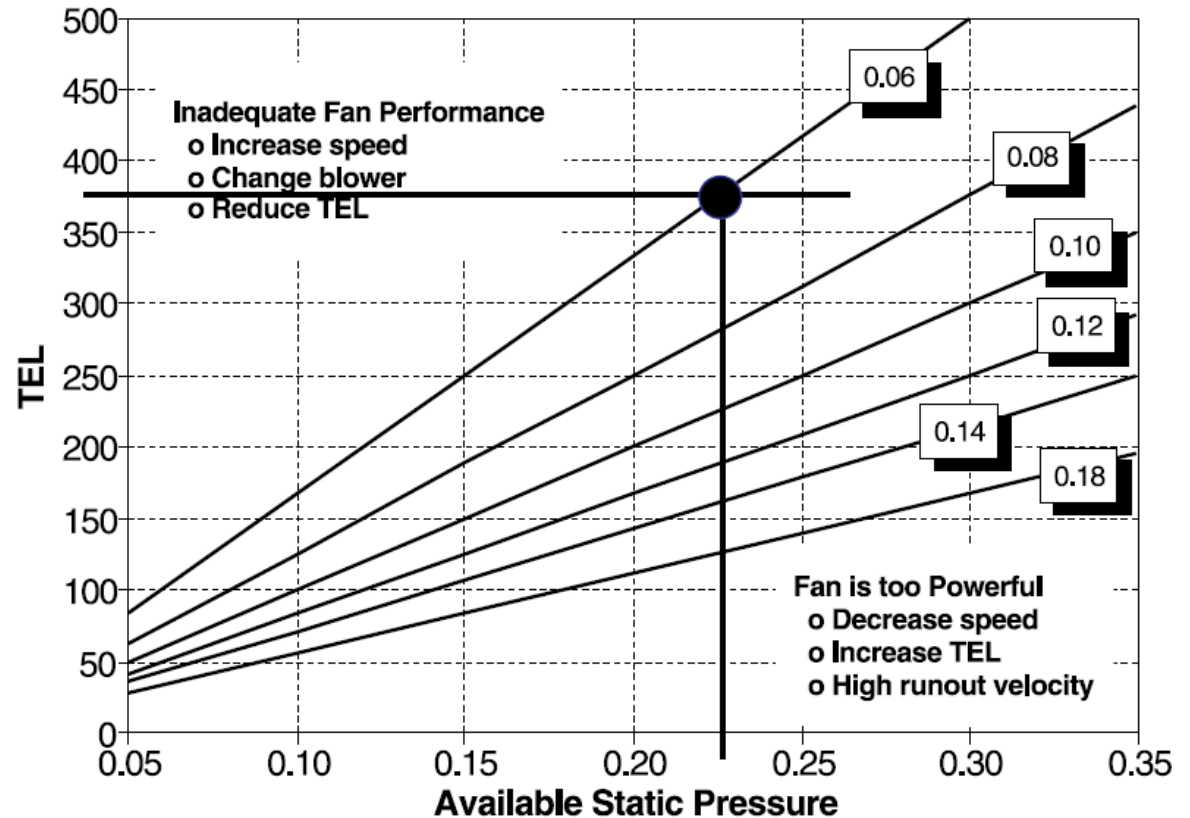
$$FR = \frac{ASP \times 100}{TEL}$$

$$FR = \frac{.23 \times 100}{375}$$

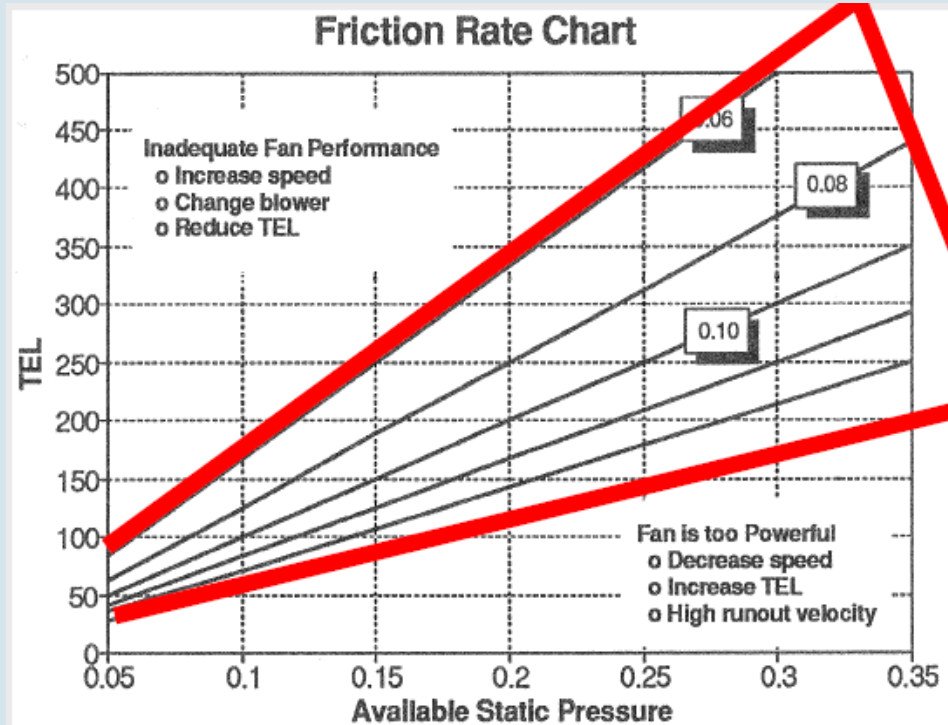
$$FR = 0.061$$

IWC / 100 Ft

### Friction Rate Chart



# Friction Rate Chart



*Outside of the “wedge” may lead to velocity problems*

# Finding Each Room Cfm

$$\text{Room CFM} = \text{Blower CFM} * \frac{\text{MJ Room Load}}{\text{MJ Total Load (htg or clg)}}$$

- One value for cooling and one value for heating
- The designer must use the larger of the two cfm values for sizing the duct runs

Reminder: Loads are in Btu/hr

# Example

- Air handler delivers 1000 Cfm at 0.23 IWC (net)
- Total heating load: 60,000 Btu/h
- Total cooling load: 48,000 Btu/h

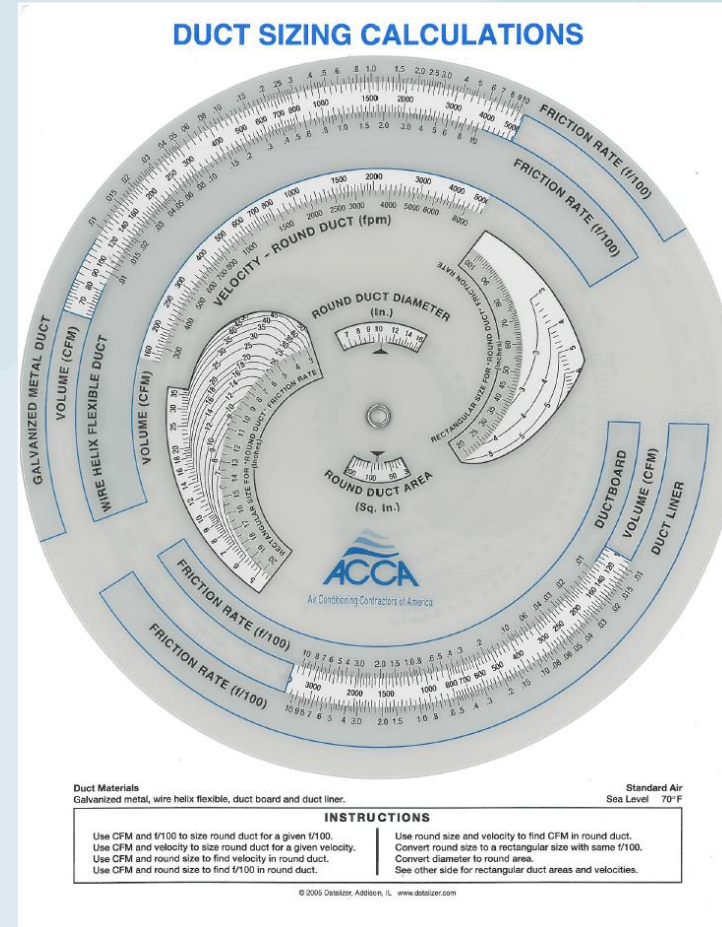
$$\text{Room CFM} = \frac{\text{Blower CFM} \times \text{MJ Room Load}}{\text{MJ Total Load (htg or clg)}}$$

Blower Cfm = 1000					
Total heating load = 60,000 Btu/h					
Total cooling load = 48,000 Btu/h					
	C - Btu/h	H - Btu/h	C - Cfm	H - Cfm	Design Cfm
Room 1	4800	5800	100	97	100
Room 2	19200	25200	400	420	420
Room 3	24000	29000	500	483	500



# FR & Cfm → Duct Size & Velocity

- Using a duct slide rule, the Cfm and calculated FR will:
  - Provide values for sizing the ducts
    - Round
    - Rectangular
  - Provide an associated velocity in feet per minute (fpm)



# Velocity Limit

- Compare the velocity (feet per minute, fpm) at the design cfm with the limits for turbulence / noise control
- If the velocity exceeds the limits, then use the cfm for the limit velocity – resulting in bigger diameter ducts

Air Velocity for Noise Control <small>Subject to Notes 1, 2 and 8</small>								
Component	Supply Side (Fpm)				Return Side (Fpm)			
	Conservative		Maximum		Conservative		Maximum	
	Rigid	Flex	Rigid	Flex	Rigid	Flex	Rigid	Flex
Trunk Ducts	700	700	900	900	600	600	700	700
Branch Ducts	600	700	900	900	500	600	700	700
Supply Outlet Face Velocity	Size for Throw		700 <small>Note 7</small>		—		—	
Return Grille Face Velocity	—		—		—		500	
Filter Grille Face Velocity	—		—		—		300	

# Manual D

## Min. Verification Points

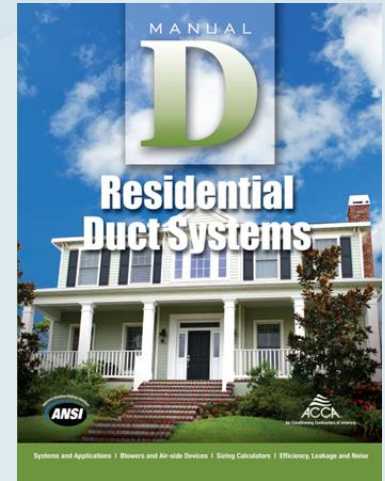


### **ACCA recommended minimum:**

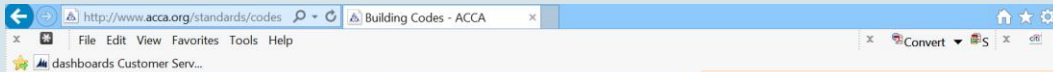
- ESP from blower table at Design Airflow (CFM)
- Total Component Pressure Losses (CPL)
- Available static pressure ( $ASP = ESP - CPL$ )
- Lengths: longest supply duct, longest return duct, TEL
- Determined Friction Rate
- Used Manual J room loads to determine Heating/Cooling CFMs
- Ensure maximum airflow velocity limits are not exceeded

# What to Watch Out For ...

- **Designers that ALWAYS use a FR of 0.10**
  - It needs to be calculated every time for the specific duct system details
- **Check the math**
  - $ASP = ESP - CPL$
  - $FR = (ASP \times 100) / TEL$
  - Spot check a few register CFMs
- **Not using balancing hand dampers in the runout branches**
- **Not altering the design for a house plan that is rotated to the opposite street side**



# Part 4: ACCA-Available Resources



Get involved in ACCA's Codes Committee and make a difference in the development and adoption of good building codes. For more info about becoming involved in the Codes Committee and ACCA's building code efforts, contact [codes@acca.org](mailto:codes@acca.org)

## Information for Code Officials

### Brochures for Code Officials

ACCA has developed several brochures which help code officials verify residential load calculations, duct design and equipment selection in accordance with Manuals J, D, and S. While it's not practical for code officials to verify every single aspect of these submissions, these brochures offer checklists for a simplified verification process. [Download them.](#)

### Load Calculation Software

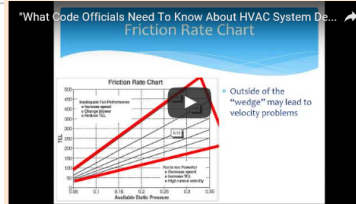
An important warning notice for code officials about the dangers of inappropriate load calculation software. [Download it.](#)

### ICC PMG Membership Council

Visit the International Code Council's (ICC) Plumbing Mechanical Gas (PMG) Membership Council webpage for other resources like CodeNotes, High School Technical Training Program Toolkit, and technical partners information.

## Video Training for Code Officials

ACCA has developed a three-part video series that aims to help code officials better understand the three main aspects of a proper residential HVAC system design: a load calculation, selecting the appropriate equipment, and proper duct sizing. The basis for the videos are the code-references ACCA Manual J, Manual S, and Manual D. The videos do not comprise a design course, but instead provide an overview of the design process and presents ACCA-recommended verification points. This will better enable code officials to verify that a system was designed correctly.



## Looking for CEUs?

ACCA is now an ICC Preferred Education Provider. Earn .02 CEUs by passing a 30 question online exam and earn your course certificate. [Click here to learn more and sign up.](#)

## ACCA in Building Codes

For years, ACCA's technical manuals and standards have been an integral part of the national model building codes' requirements for proper HVAC design. The following model codes currently reference, or have in the past referenced, ACCA's various design manuals and standards:

- IAPMO's Uniform Mechanical Code
- IAPMO's Uniform Swimming Pool, Spa, and Hot Tub Code
- ICC's International Residential Code
- ICC's International Energy Conservation Code
- ICC's International Mechanical Code

To see the detailed references in each model code, including section excerpts, please see the document for the specific cycle:

- [2015 Model Code References](#)
- [2012 Model Code References](#)
- [2009 Model Code References](#)

[www.acca.org/codes](http://www.acca.org/codes)



# Residential Plans Examiner Review Form for HVAC System Design (Loads, Equipment, Ducts)

Form  
RPER 1.01  
8 Mar 10

County, Town, Municipality, Jurisdiction  
Header Information

Contractor \_\_\_\_\_  
Mechanical License # \_\_\_\_\_  
Building Plan # \_\_\_\_\_  
Home Address (Street or Lot#, Block, Subdivision) \_\_\_\_\_

**REQUIRED ATTACHMENTS<sup>1</sup>**  
Manual J1 Form (and supporting worksheets):  
or MJ1AE Form<sup>2</sup> (and supporting worksheets):  
OEM performance data (heating, cooling, blower):  
Manual D Friction Rate Worksheet:  
Duct distribution system sketch:

**ATTACHED**  
Yes  No   
Yes  No   
Yes  No   
Yes  No   
Yes  No

## HVAC LOAD CALCULATION (IRC M1401.3)

### Design Conditions

#### Winter Design Conditions

Outdoor temperature \_\_\_\_\_ °F  
Indoor temperature \_\_\_\_\_ °F  
Total heat loss \_\_\_\_\_ Btu

#### Summer Design Conditions

Outdoor temperature \_\_\_\_\_ °F  
Indoor temperature \_\_\_\_\_ °F  
Grains difference \_\_\_\_\_ Δ Gr @ \_\_\_\_\_ % Rh  
Sensible heat gain \_\_\_\_\_ Btu  
Latent heat gain \_\_\_\_\_ Btu  
Total heat gain \_\_\_\_\_ Btu

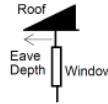
### Building Construction Information

#### Building

Orientation (Front door faces) \_\_\_\_\_  
North, East, West, South, Northeast, Northwest, Southeast, Southwest  
Number of bedrooms \_\_\_\_\_  
Conditioned floor area \_\_\_\_\_ Sq Ft

#### Windows

Eave overhang depth \_\_\_\_\_ Ft  
Internal shade \_\_\_\_\_  
Blinds, drapes, etc \_\_\_\_\_  
Number of skylights \_\_\_\_\_



## HVAC EQUIPMENT SELECTION (IRC M1401.3)

### Heating Equipment Data

Equipment type \_\_\_\_\_  
Furnace, Heat pump, Boiler, etc.  
Model \_\_\_\_\_  
Heating output capacity \_\_\_\_\_ Btu  
Heat pumps - capacity at winter design outdoor conditions  
Auxiliary heat output capacity \_\_\_\_\_ Btu

### Cooling Equipment Data

Equipment type \_\_\_\_\_  
Air Conditioner, Heat pump, etc.  
Model \_\_\_\_\_  
Sensible cooling capacity \_\_\_\_\_ Btu  
Latent cooling capacity \_\_\_\_\_ Btu  
Total cooling capacity \_\_\_\_\_ Btu

### Blower Data

Heating CFM \_\_\_\_\_ CFM  
Cooling CFM \_\_\_\_\_ CFM

## HVAC DUCT DISTRIBUTION SYSTEM DESIGN (IRC M1601.1)

Design airflow \_\_\_\_\_ CFM  
External Static Pressure (ESP) \_\_\_\_\_ IWC  
Component Pressure Losses (CPL) \_\_\_\_\_ IWC  
Available Static Pressure (ASP) \_\_\_\_\_ IWC  
ASP = ESP - CPL  
Longest supply duct: \_\_\_\_\_ Ft  
Longest return duct: \_\_\_\_\_ Ft  
Total Effective Length (TEL) \_\_\_\_\_ Ft  
Friction Rate: \_\_\_\_\_ IWC  
Friction Rate = (ASP × 100) ÷ TEL  
Duct Materials Used (circle)  
Trunk Duct: Duct board, Flex, Sheet metal, Lined sheet metal, Other (specify) \_\_\_\_\_  
Branch Duct: Duct board, Flex, Sheet metal, Lined sheet metal, Other (specify) \_\_\_\_\_

I declare the load calculation, equipment selection, and duct system design were rigorously performed based on the building plan listed above, I understand the claims made on these forms will be subject to review and verification.

Contractor's Printed Name \_\_\_\_\_ Date \_\_\_\_\_  
Contractor's Signature \_\_\_\_\_

Reserved for use by County, Town, Municipality, or Authority having jurisdiction.

<sup>1</sup> The AHJ shall have the discretion to accept Required Attachments printed from approved ACCA software vendors, see list on page 2 of instructions.

<sup>2</sup> If abridged version of Manual J is used for load calculation, then verify residence meets requirements, see Abridged Edition Checklist on page 13 of instructions.

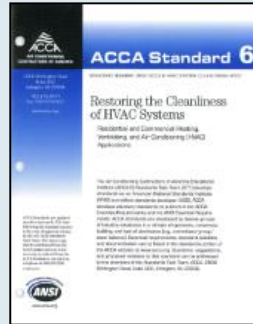
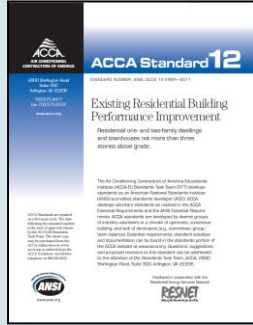
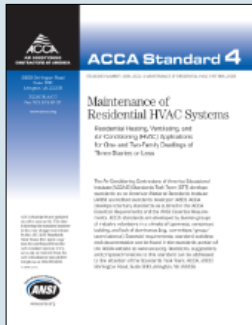
# ACCA Design Review Form

## Everything you need to check on one form.

- Load calculation
- Equipment selection
- Duct system design

Free to download at  
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# Free Standards



## Free PDF Downloads on HVAC

- Quality Installation (ACCA 5 QI)
- QI Verification (ACCA 9 QIvp)
- Quality Maintenance (ACCA 4 QM)
- Quality Restoration (ACCA 6 QR)
- Whole House Evaluation (ACCA 12 QH)
- and more

Free to download at [www.acca.org/quality](http://www.acca.org/quality)

# Free Training for Code Officials (and Others!)



## Three-part video training on Manuals J / D / S

- Approximately 45 minutes for each segment
- A bit more detailed than this presentation
- Free! ... [www.ACCA.org/codes](http://www.ACCA.org/codes)

## CEUs available from ICC

- ACCA is an ICC Preferred Education Provider
- [See: http://www.acca.org/certification/code-essentials](http://www.acca.org/certification/code-essentials)
- 0.2 CEU; Cost for the J / D / S test = \$60

CEUs have  
associated  
costs.



# ACCA Technical Reference Note

*“Computing Manual J Infiltration Load Based Upon a Target Envelop Leakage Requirement”*



Shows how to convert a maximum code allowable leakage limit (say, 3 or 5 ACH 50 per the ICC IECC) to:

1. Manual J infiltration CFM value, and then to
2. infiltration load contributions (Btuh) of:
  - sensible heating,
  - sensible cooling, and
  - latent cooling.

# Free ACCA Membership for ICC Code Offices



**To obtain ACCA member benefits for free,  
contact:**

Karla Price Higgs  
Vice President, Member Services  
International Code Council  
[KHiggs@iccsafe.org](mailto:KHiggs@iccsafe.org)

# Educational Offerings

## QI Design ... [Load Calcs, Equipment Selection, Duct Design, etc.]



### Offered via:

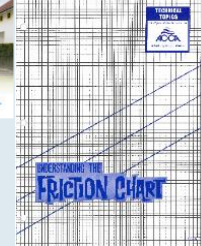
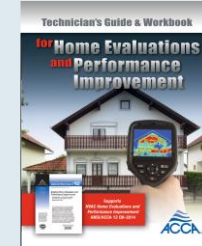
- In-person training (3-day class)
- Online training (18 hours of videos, plus assessments)
- Offline DVDs

# Educational: Technician Training & Certification



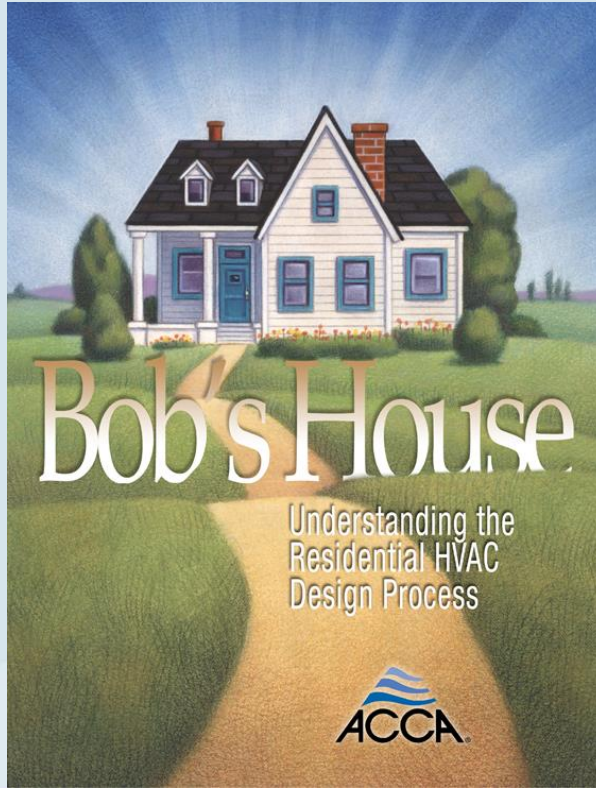
## On-line learning

- *Technician Field Practices for Quality Installation*
- *Home Evaluation and Performance Improvement*
- *Friction Rate Primer and Duct Design Fundamentals*
- *Duct Diagnostics & Repair*
- *Etc.*



Convenient ... affordable ... on-demand training focused on quality HVACR installation, maintenance, home performance, and more.

These each have associated costs.

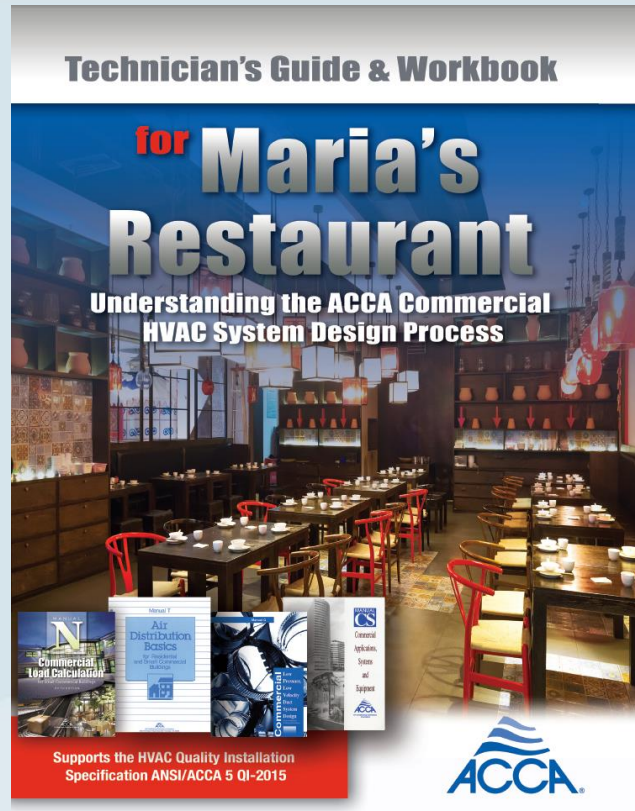


## Bob's House

A case study for understanding the residential HVAC design process as described in the ACCA residential design manuals.

May be  
purchased at

[www.acca.org/store/](http://www.acca.org/store/)



## Maria's Restaurant

A case study for understanding the commercial HVAC design process as described in the ACCA commercial design manuals.

May be purchased at

[www.acca.org/store/](http://www.acca.org/store/)

Questions ???

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